



AD FALCON API Manual

Validation of FEM Code with an Axisymmetric Foundation Model Using a Linear Elastic Material

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1 Validation of FEM Code with an Axisymmetric Foundation Model Using a Linear Elastic Material

1.1 File Name

fem_2_axisymmetric.txt

1.2 Problem Description

This example builds on [Example 1](#) but analyzes the foundation model under **axisymmetric conditions**. The geometry, material properties, and loading conditions remain the same as described in [Example 1](#). Static mode is used alongside an autoincrement option.

The objective is to validate the vertical stress (σ_{yy}) distribution predicted by the FEM code against the analytical solution derived from the Boussinesq equations, using axisymmetric mechanics.

Refer to [Example 1](#) for the geometry and boundary conditions.

1.3 FEM Model Setup

- **Analysis Type:** Axisymmetric Non-coupled (AXUnCoupled). Element Type, Material Properties, Step Definition and Load Configuration are the same as [Example 1](#)

1.4 Results

The FEM solution for the vertical stress (σ_{yy}) distribution under axisymmetric conditions is compared to the analytical solution along the depth beneath the loaded area.

1.4.1 Comparison with Analytical Solution

The figure below shows the plot of σ_{yy} versus depth:

Figure 1. Predictions against analytical solution beneath the center of the footing under axisymmetric conditions.

1.5 Analytical Settlement at the Center (Flexible Circular Load)

For a uniformly loaded circular area on a linear-elastic, homogeneous, isotropic half-space (flexible load), the maximum surface settlement at the center is

$$w_{\max}(r = 0) = \frac{2(1 - \nu^2)}{E} q a$$

where q is the uniform pressure, a is the load radius, E is Young's modulus, and ν is Poisson's ratio.

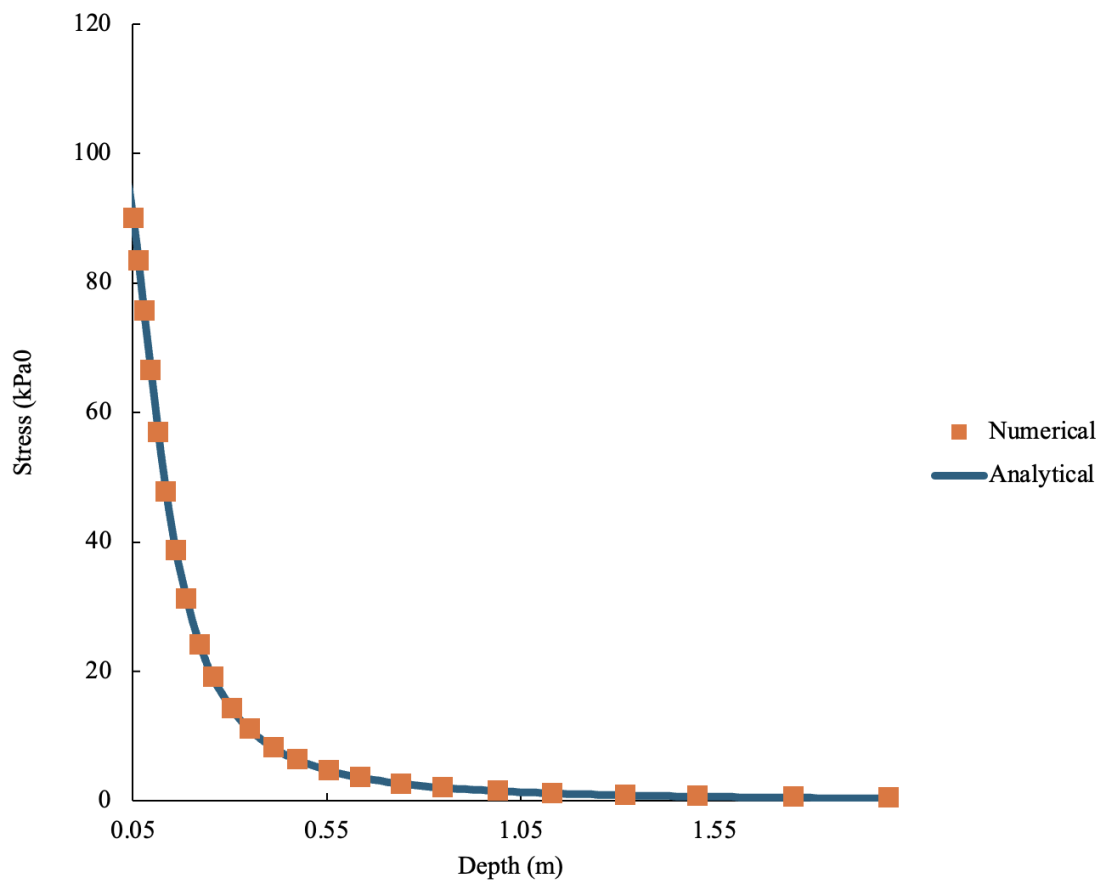


Figure 1: Predictions against analytical solution beneath the center of the footing.

Example with the properties used here: $E = 210,000 \text{ kPa}$, $\nu = 0.3 \Rightarrow (1 - \nu^2) = 0.91$, $q = 100 \text{ kPa}$, $a = 0.10 \text{ m}$:

$$w_{\max} = \frac{2 \times 0.91}{210000} \times 100 \times 0.10 = 8.6667 \times 10^{-5} \text{ m} = 0.08667 \text{ mm}$$

1.5.1 Why the FEM settlement is slightly smaller

- Finite domain and rigid boundaries (box 2×2 m with $u_y = 0$ at the base) limit deep strains compared to an infinite half-space.
- Lateral restraints ($u_x = 0$ at vertical sides) restrict spreading beneath the load, increasing apparent stiffness.
- Minor discretization and element-formulation effects.

Comparison at the center: analytical $w_{\max} = 0.08667 \text{ mm}$ vs FEM $w_{\max} = 0.0835 \text{ mm} \rightarrow$ about -3.65% (FEM smaller for the reasons above).

In the provided input file (`fem_2_axisymmetric.txt`), the FEM settlement at the center is recorded via `% DOFOutput to dof_center_disy_axisymmetric.csv`.

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